Leveraging research in computational mechanics to improve undergraduate teaching

Julián J. Rímoli

School of Aerospace Engineering

Georgia Institute of Technology

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Before we start:

• Download Truss Me! (freely available)



Google Play

Truss Me! scientific Monkey LLC Educational Everyone A You don't have any devices.



Apple's App Store







Outline

- Part 1: Tensegrity structures for planetary landers.
 - Motivation
 - Model development
 - Impact behavior
- Part 2: Truss Me! A game-based learning tool for structural mechanics.
 - Introduction
 - Hands-on activities





Part 1

Tensegrity planetary landers



Planetary landing structures 101



Worst-case scenario: impact without dissipation $\implies \frac{1}{2}mv^2 = \frac{1}{2}k\delta^2$

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 'turn EXIT_SUCCESS)



Background

- SunSpiral first proposed the use of tensegrity structures for planetary landers [1].
- Solution inspired by tensegrity toy's ability to recover from extreme deformation.
- Main idea:
 - Structure composed of rigid bars and elastic cables
 - Generate locomotion by actuating cable members
- Main limitation:
 - Assumed rigid bars, and all energy absorbed by cables
 - This would lead to mass that does not contribute to energy storage!

[1] V. SunSpiral et al, International Journal of Planetary Probes, 2013







Tensegrity Planetary Lander: our concept

- Design a tensegrity structure that exploits elasticity of bars.
- Tensegrity octahedron's super-stable property [1] makes it a good candidate.
- Payload located at the center (cable suspended)
- Main advantages:
 - Higher impact velocity (or higher payload)
 - Strain energy evenly distributed throughout the structure.

[1] Zhang et al, International Journal of Solids and Structures, 2019



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A distinctive design approach: Let it buckle

- We propose to design tensegrity structures whose bar members are allowed to buckle in the elastic regime.
- This will bring several benefits (to be discussed later in this presentation) for impact problems.
- From a design perspective, a bar can be easily dimensioned to ensure elastic buckling:

$$\frac{L}{\rho} \ge \sqrt{\frac{2\pi^2 E}{\sigma_y}}$$



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Post-buckling behavior

- When loading a simply supported bar axially, the response is linear up to the buckling point.
- Consequently, the stored strain energy is quadratic. This is the amount of energy we would be able to store if buckling is not allowed.
- Beyond buckling, the load does not drop. On the contrary, it remains almost constant (slightly increases)
- Since relatively large displacements can be applied after buckling, large amounts of additional strain energy can be stored on the bar.
- In this example, a 58cm long Ti tubular bar with 19mm diameter and 1mm thickness can store 58.5 J in the post-buckling regime vs 2.1 J if buckling is not allowed (27x increase in this case)



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Modeling tensegrity structures



Contents lists available at ScienceDirect

Mechanics of Materials

journal homepage: www.elsevier.com/locate/mechmat

A reduced-order model for the dynamic and post-buckling behavior of tensegrity structures

Julian J. Rimoli

School of Aerospace Engineering, Georgia Institute of Technology, Atlanta, GA 30332, USA





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Modeling tensegrity structures



- The continuum bar is discretized as a symmetric system with 4 masses, 3 axial springs and 2 angular springs.
- We must determine all stiffnesses and masses to reproduce required bar properties.
- Computation of stiffness of axial springs is trivial.
- The distance between the two interior masses is left as a discretization parameter ($h=\alpha L$)

Rimoli, MOMS, 2018





Mass properties



• By forcing the discrete system to have the same mass and mass moment of inertia as the continuum bar we obtain:

$$2m_1 + 2m_2 = \rho AL$$

$$\frac{1}{2}m_1L^2 + \frac{1}{2}m_2\alpha^2L^2 = \frac{1}{12}\rho AL^3$$

$$m_1 = \frac{1}{6}\rho AL\left(\frac{1-3\alpha^2}{1-\alpha^2}\right)$$

$$m_2 = \frac{1}{3}\rho AL\left(\frac{1}{1-\alpha^2}\right)$$

- To have positive mass, the previous expressions impose a limit on the value of $\boldsymbol{\alpha}$

 $0 \leq \alpha \leq \sqrt{3}/3$

Rimoli, MOMS, 2018





Buckling load

- Let us consider the discretized system in the buckled configuration.
- The potential energy of the system is given by

$$\Pi = k_1 \left(d_1 - \frac{L-h}{2} \right)^2 + \frac{k_2}{2} \left(d_2 - h \right)^2 + k_t \theta^2 - P \Delta L$$

• Where d_1 , d_2 , and θ are the generalized coordinates of the problem, and the displacement at the end of the bar is given by

 $\Delta L = L - 2d_1\cos(\theta) - d_2$

• The non-trivial solution gives us: $P_{cr} = \frac{1}{4}k_1(L-h) - \frac{1}{4}\sqrt{k_1^2(L-h)^2 - 16k_1k_t}$



Buckling load



 Equating the critical buckling load for the discrete and continuum systems, and solving for k_t we get

$$k_t = \frac{1-\alpha}{2} \frac{\pi^2 EI}{L} \left(1 - \pi^2 \frac{I}{AL^2} \right)$$

Rimoli, MOMS, 2018



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Discretized system



$$m_1 = \frac{1}{6}\rho AL\left(\frac{1-3\alpha^2}{1-\alpha^2}\right)$$
$$m_2 = \frac{1}{3}\rho AL\left(\frac{1}{1-\alpha^2}\right)$$
$$h_1 = \frac{1-\alpha}{3}\frac{\pi^2 EI}{1-\alpha^2}\left(1-\pi^2\right)$$

$$k_t = \frac{1-\alpha}{2} \frac{\pi^2 EI}{L} \left(1 - \pi^2 \frac{I}{AL^2} \right)$$

- The discretized system has the following properties:
 - Same mass as continuum bar
 - Same mass moment of inertia as the continuum bar
 - Same axial stiffness as the continuum bar
 - Same buckling load as the continuum bar
 - Nearly same post-buckling load as the continuum bar
 - Low error on **first two natural frequencies**, with second frequency exactly twice the first one (with $\alpha = 1/2$)

With only **12 DOFs per** bar we can study dynamic and post-

 buckling behavior of truss and tensegrity structures/lattices

Rimoli, MOMS, 2018



Implementation details



Rimoli, MOMS, 2018



Implementation details

```
// Compute unit vectors
for (int i = 0; i < _n_segments; i++){</pre>
    int n1 = _connectivity[i];
    int n2 = _connectivity[i+1];
    _t[i] = points[n2]._position - points[n1]._position;
    _d[i] = _t[i].magnitude;
    _u[i] = _t[i]/_d[i];
// Set forces to zero
for(int i=0; i < _n_segments+1; i++){</pre>
    F[i].Set(0,0,0);
}
if(elType == "Bar"){
// Compute angular spring forces
    float cost, theta, theta2, theta4, theta6, theta8, coef;
    Vector3 sint, f1, f2, f3;
    for (int i = 0; i< _n_int_points; i++){</pre>
        sint = Vector3.Cross(_u[i],_u[i+1]);
        cost = Vector3.Dot(_u[i],_u[i+1]);
        theta = Mathf.Atan(sint.magnitude/cost);
        theta2 = theta * theta;
        theta4 = theta2*theta2;
        theta6 = theta2*theta4:
        theta8 = theta4*theta4;
        coef = kt[i]/(1f-theta2/6f+theta4/120f-theta6/5040f+theta8/362880f);
        f1 = coef/_d[i]*Vector3.Cross(_u[i],sint);
        f3 = coef/_d[i+1]*Vector3.Cross(_u[i+1],sint);
        f2 = f1+f3;
        F[i] += f1;
        F[i+1] -= f2;
        F[i+2] += f3;
    3
```

```
// Compute linear spring forces
float force;
for (int i = 0; i < _n_segments ; i++){
    force = _k[i]*(_d[i]-_d0[i]);
    F[i] += force * _u[i];
    F[i+1] -= force * _u[i];
}</pre>
```

```
// Assemble forces
for(int i=0; i < _n_segments+1; i++){
    points[_connectivity[i]]._force += F[i];</pre>
```

```
}
```





Tensegrity planetary lander: concept design



Rimoli, MOMS, 2018











Force history on bars







With pre-stress







Rimoli, MOMS, 2018



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06 67 return EXIT_SUCCESS;}

Maximum peak forces on bars and cables



- By allowing them to buckle, bars act as a load-limiting mechanism.
- It is not only the load that bars experience that is limited: they also limit the maximum load experienced by cables.

Rimoli, MOMS, 2018



3D tensegrity lattice



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3D tensegrity lattice



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 Georgia Institute of Technology
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Part 2

A game-based learning tool for structural mechanics





Truss mel: Combining advanced simulations and gaming for engineering education

- Designed an educational app to help students build a conceptual understanding on how truss structures behave and fail.
- Based on a **state-of-the-art simulation engine** to provide the most realistic experience.
- **Game-like approach**: students solve puzzles, of increasing difficulty, by designing load-bearing structures.
- Available on Google Play and iOS App Store
 - Over 450,000 downloads in 150 countries
- Incorporated for teaching in numerous middle and elementary schools, and some of the most prestigious engineering schools including ETH Zurich, Georgia Tech, Vanderbilt University, Rutgers, and Caltech.







Classroom implementation I:

Simulation of engineering design cycle in the classroom

Advanced Manufacturing







Testing





Classroom implementation II: Semester-long truss design competition

Rules

- By the end of the day on each dates specified in the <u>calendar at the end of this</u> <u>document</u>, students will upload to canvas a screenshot of their best design for the corresponding challenge.
- 2- For each challenge, scores from all students will be normalized by the highest score for that level. That is, the top design in the class will get 1 point.
- 3- By the end of the semester, students will get a final score consisting on the sum of their normalized scores. There is a total of 15 challenges to be solved, so the maximum possible score is 15.
- 4- The final score of each student will then be normalized by the highest final score in the class. That is, the highest possible normalized score is 1.
- 5- For each student, the contribution from this competition to the final grade will be 10% multiplied by the normalized final score. That is, the top student in the class will get 10% toward the final grade.
- 6- The top 4 students in the class will participate on a live finale on Monday, April 22nd 2019.
- 7- The format of the finale is as follows:
 - a. All 4 finalists will get the same challenge to solve, and will be given 15 minutes to get to their best design. All devices will be streamed live to the classroom screen so students in the class will be able to see the design process, and cheer for their favorite competitor if they wish!
 - b. The 2 best performers from the first round will pass to the final round, which will have the same format as the previous one.
 - c. The winner will be given a \$50 gift card, and a \$25 gift card will be awarded to the second place. Gift cards are courtesy of Prof. Rimoli ⁽²⁾

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turn EXIT_SUCCESS;}

Hands-on activities!

- Start Truss Me! on your phone or tablet
- For this activity, please wait for the instructor for each step to avoid spoilers!



Acknowledgements



The main (Int args, char 'argv[))(Computational Solid Mechanics Lab School of Aerospace Engineering Georgia Institute of Technology



Thanks!

• Questions?



